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Article

An LMI Approach to Nonlinear State-Feedback Stability of Uncertain Time-Delay Systems in the Presence of Lipschitzian Nonlinearities

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Abstract: This article proposes a new nonlinear state-feedback stability controller utilizing linear matrix inequality (LMI) for time-delay nonlinear systems in the presence of Lipschitz nonlinearities and subject to parametric uncertainties. Following the Lyapunov–Krasovskii stabilization scheme, the asymptotic stability criterion resulted in the LMI form and the nonlinear state-feedback control technique was determined. Due to their significant contributions to the system stability, time delays and system uncertainties were taken into account while the suggested scheme was designed so that the system’s stabilization was satisfied in spite of time delays and system uncertainties. The benefit of the proposed method is that not only is the control scheme independent of the system order, but it is also fairly simple. Hence, there is no complexity in using the proposed technique. Finally, to justify the proficiency and performance of the suggested technique, a numerical system and a rotational inverted pendulum were studied. Numerical simulations and experimental achievements prove the efficiency of the suggested control technique.

Keywords: state-feedback stabilization; linear matrix inequality; Lipschitz nonlinearity; parametric uncertainty; time delays

1. Introduction

Time delays are usually encountered in numerous industrial systems that must be controlled, such as distributed networks, chemical processes, telecommunications, electrical servo systems, and nuclear reactors, etc. [1–4]. Because states of the system depend on the present time and a time period in the past, time delays occur in dynamical systems, and ignoring their effect yields severe deterioration in system performance or even system instability [5,6]. Thus, control of time-delay systems is known to have practical significance [7–9]. Recent decades have observed a widespread attention given to the synthesis of appropriate control laws for time-delay dynamical systems in the presence of parameter uncertainties [10–12]. In [13], an adaptive fuzzy backstepping technique was proposed for the nonlinear dynamical systems with unmeasured states and unknown time delays. In [14], H_∞ stabilization control for the time-delay Takagi–Sugeno fuzzy systems under nonlinear perturbations and sampled-data input was investigated. By using output feedback, the authors of [15] studied the robust stabilization problem of a class of time-varying time-delay dynamical systems, which were not perfectly known, where the system output was modeled through a nonlinear function depending on the delayed inputs and states. In [16], the state-feedback stability control of switched

discrete-time singular systems in the presence of time-varying state delays was presented. In [17], the stabilization problem of nonlinear cascade time-delay systems using the converse Lyapunov stabilization and invariant set theories was presented.

From a practical point of view, most process models, including power systems [18], robotic manipulators [19], non-holonomic systems [20], and flexible space structures [21], suffer from system uncertainties. Thus, system uncertainties should always be taken into account when a control system for both stability and performance is designed. The design of a robust nonlinear state-feedback control scheme that overcomes system uncertainties has been the subject of substantial investigation over the years. The linear matrix inequality (LMI) approach is a suitable and strong technique to deal with system uncertainties including parametric [22] or unstructured uncertainties [23]. Due to its influential structure, the LMI technique has been widely applied to obtain solutions for convex problem minimization, such as H_∞ control [24] and H_2 control [25]. Dealing with difficult problems for which there is no analytical solution is another significant feature of the LMI approach that attracts the attention of many researchers since it offers numerically tractable means [26,27]. Furthermore, powerful algorithms exist, such as interior-point ones, to provide a way of dealing with LMI problems. Two robust H_∞ state-feedback controllers based on LMIs for time-delay discrete-time systems and uncertain switched impulsive linear systems were proposed in [28] and [29], respectively. In [30], a robust H_∞ fuzzy-logic controller for Takagi–Sugeno time-delay bilinear discrete-time systems in the presence of disturbances was presented in which the stabilization conditions were formulated as LMI. A combination of Lyapunov parameter-dependent function and LMI was also used in [31,32] to develop a control scheme for uncertain systems subject to time delay. Moreover, based on the LMI approach, the problem of stabilization of a uniform Euler–Bernoulli beam and a two-dimensional Burgers' equation have been investigated in [33] and [34], respectively. To the best of the authors' knowledge, little consideration has been drawn to the problem of nonlinear state-feedback stability for nonlinear time-delay systems with Lipschitz nonlinearities via LMIs.

This work aimed to present a state-feedback controller for the stabilization of Lipschitz nonlinear systems. Time delays and parametric uncertainties were also taken into account due to their significant contribution to the system stability. Using a Lyapunov–Krasovskii functional, some asymptotic stability conditions were formed as LMI and the parameters of the state-feedback controller were found through LMI. The offered control law ensures asymptotic stabilization of the systems, even if the nonlinear function is not equal to zero. Unlike the previous investigations, the LMI conditions possess fewer pre-assumed design parameters, and thus, the designed method may lead to less conservative conditions. Moreover, the control scheme is independent of the order of systems' model. The chief novelties of the planned technique are presented as follows:

- Design of a nonlinear state-feedback stabilizing controller for nonlinear systems in the presence of time delays, Lipschitz nonlinearity, and parametric uncertainties.
- Achievement of asymptotic stabilization based on the Lyapunov–Krasovskii stabilization theory and the LMI approach.
- The suggested control scheme is rather straightforward; there is no difficulty in the employment of this technique.
- Application of the offered method on a nonlinear unstable system and a rotational inverted pendulum to prove the efficiency of the method.

The presentation of this work is as follows: Section 2 gives the definition of the problem and required preliminaries. Section 3 presents the stability analysis and design process of the LMI-based nonlinear state-feedback control law for the time-delay systems in the presence of uncertainties and nonlinearities. In Section 4, some simulation and experimental results are illustrated. Finally, Section 5 concludes the paper.

2. Problem Description

Consider the uncertain time-delay system as

$$\begin{aligned}\dot{x}(t) &= f(x) + (A + \Delta A)x(t) + A_1x(t - \tau) + (B + \Delta B)u(t), \\ y(t) &= Cx(t),\end{aligned}\quad (1)$$

where $\tau \in R$, $u(t) \in R^n$, $y(t) \in R^p$, and $x(t) \in R^n$ represent the time delay, control inputs, outputs, and states of the system, respectively. The matrices A , A_1 , B , and C are constant matrices with suitable dimensions; the nonlinear function $f(x) \in R^n$ is a time-varying vector; ΔA and ΔB signify the parametric uncertainties.

Lemma 1. For any real $\gamma > 0$, the following inequality is satisfied [35]:

$$A^T B + B^T A \leq \gamma A^T A + \frac{1}{\gamma} B^T B \quad (2)$$

where the matrices A and B are of consistent dimensions.

Lemma 2 (Schur Complement) [36]. Consider the symmetric block matrix $\Pi = \begin{bmatrix} A & B \\ B^T & D \end{bmatrix}$, where $A = A^T$ and $D = D^T$, the condition $\Pi < 0$ is equivalent to $D < 0$ and $\Gamma_D = A - BD^{-1}B^T < 0$. Similarly, the condition $\Pi < 0$ is equivalent to $A < 0$ and $\Gamma_A = D - B^T A^{-1}B < 0$.

Assumption 1. $f(x)$ is a Lipschitzian nonlinear function for all $x \in R^n$ and $\bar{x} \in R^n$ satisfying [37]:

$$\|f(x) - f(\bar{x})\| \leq L\|x - \bar{x}\|, \quad (3)$$

where $L \in R^{n \times n}$ is a constant Lipschitz matrix. The inequality (3) can be written as

$$(f(x) - f(0))^T I (f(x) - f(0)) \leq x^T L^T L x. \quad (4)$$

The state-feedback controller law is designed as

$$u(t) = Fx(t) - B^{-1}f(0), \quad (5)$$

where F denotes the state-feedback gain and additional term $B^{-1}f(0)$ is necessary to deal with systems possessing $f(0) \neq 0$.

Remark 1. If B is a non-square and full-rank matrix, the state-feedback controller is designed utilizing the right inverse of B (pseudo-inverse) as:

$$u(t) = Fx(t) - B^T (BB^T)^{-1} f(0). \quad (6)$$

3. Nonlinear State-Feedback Stabilization

In the subsequent theorem, a sufficient condition for the stability of system (1) is presented.

Theorem 1. Consider the uncertain nonlinear time-delay system (1) and the controller input (5). If there exist matrices $Q = Q^T > 0$, Y , and H with suitable dimensions so that the LMI holds as:

$$\Gamma = \begin{bmatrix} \Omega & A_1 Q & I & 0 & D_A & D_B & Q \\ * & H & 0 & 0 & 0 & 0 & 0 \\ * & * & -I & 0 & 0 & 0 & 0 \\ * & * & * & \gamma_C B^{-T} E_B^T E_B B^{-1} & 0 & 0 & 0 \\ * & * & * & * & -\gamma_A I & 0 & 0 \\ * & * & * & * & * & -\frac{\gamma_B \gamma_C}{\gamma_B + \gamma_C} & 0 \\ * & * & * & * & * & * & -M \end{bmatrix} < 0, \quad (7)$$

where $\Omega \triangleq A Q + Q A^T + B Y + Y^T B^T + \gamma_B Y^T E_B^T E_B Y - H$, $M = (L^T L + \gamma_A E_A^T E_A)^{-1}$, the parameters γ_A , γ_B and γ_C are positive constants, D_A , D_B , E_A , and E_B are known constant matrices representing system uncertainties, and $P = Q^{-1}$ and $P_1 = Q^{-1} H Q^{-1}$ are positive-definite matrices, then the control signal (5) guarantees the asymptotical stability of the system states and one can calculate F in (5) as $F = Y Q^{-1}$.

Proof. The proof of this theorem is provided in Appendix A. \square

Remark 2. The notations “*”, “<”, and “ \triangleq ” denote the transpose conjugate matrix of a symmetric matrix to main diagonal, the negativeness of the real parts of all eigenvalues of the matrix, and the equivalent equality, respectively.

4. Simulation and Experimental Results

In this section, two instances are considered in order to investigate the performance of the suggested method. In the first example, an unstable nonlinear system with state delay is studied and an LMI-based controller is developed to overcome the time delay, nonlinearity, and uncertainties of the system. In the second example, the offered control approach is applied to a practical rotary inverted pendulum (RIP) system with state-delayed and nonlinear terms.

4.1. Example A: Unstable Nonlinear System

The equations of an unstable nonlinear numerical system are given by [1]:

$$\begin{aligned} \dot{x} &= \begin{pmatrix} 0.2 \cos x_1 \\ 0.3 \sqrt{x_2^2 + 5} \\ 0.4 \sin x_3 \end{pmatrix} + \begin{pmatrix} 0 & 0.01 & -0.01 \\ 0.05 & -0.04 & 0 \\ 0.01 & 0.02 & -0.03 \end{pmatrix} x(t - \tau) \\ &+ \left[\begin{pmatrix} 0.1 & 0.2 & 0.3 \\ 0.1 & 0.1 & -0.5 \\ 0.3 & -0.4 & -0.3 \end{pmatrix} + \begin{pmatrix} 0.027 & 0.0345 & 0.038 \\ 0.02 & 0.017 & 0.034 \\ 0.0335 & 0.0325 & 0.0455 \end{pmatrix} \sin(3t) \right] x \\ &+ \left[\begin{pmatrix} 0.012 & 0.013 & 0.014 & 0.016 \\ 0.01 & 0.014 & 0.01 & 0 \\ 0.013 & 0.017 & 0.018 & 0.011 \end{pmatrix} + \begin{pmatrix} 0.0024 & 0.0026 & 0.0028 & 0.0032 \\ 0.002 & 0.0028 & 0.002 & 0 \\ 0.0026 & 0.0034 & 0.0036 & 0.0022 \end{pmatrix} \cos(2t) \right] u, \\ y &= \begin{pmatrix} 1.5 & 2 & 1.25 \\ 0.84 & 0.5 & 0.2 \end{pmatrix} x. \end{aligned} \quad (8)$$

Suppose that the time delay and initial conditions of the system are respectively taken as $\tau = 2$, $x(0) = [104 - 6]^T$. The LMI condition (7) is solved by MATLAB[®] YALMIP[®] solver as:

$$H = \begin{bmatrix} 0.086 & 0.037 & 0.03 \\ 0.037 & 0.182 & 0.042 \\ 0.03 & 0.042 & 0.112 \end{bmatrix}, P = \begin{bmatrix} 20.63 & -11.27 & -12.42 \\ -11.27 & 33.16 & -6.02 \\ -12.42 & -6.02 & 55.46 \end{bmatrix}, P_1 = \begin{bmatrix} 51.11 & -48.67 & -72.99 \\ -48.67 & 130.36 & -3.55 \\ -72.99 & -3.55 & 298.24 \end{bmatrix}, F = \begin{bmatrix} -95.17 & 39.47 & 205.77 \\ -57 & -86.44 & -53.18 \\ 105.91 & 46.86 & -233.61 \\ -29.98 & 92.37 & 93.68 \end{bmatrix}$$

Simulations of the proposed controller are given in Figures 1 and 2, as compared to the simulations of [1]. For the comparison of the simulation outcomes with those of [1], system (8) is also simulated via the control input described in [1]. Figure 1 illustrates the states of system (8), which proves that the proposed control method displays faster and superior responses over those of [1]. As shown in Figure 1, the proposed method results in a more acceptable steady-state behavior of the system states in terms of accuracy.

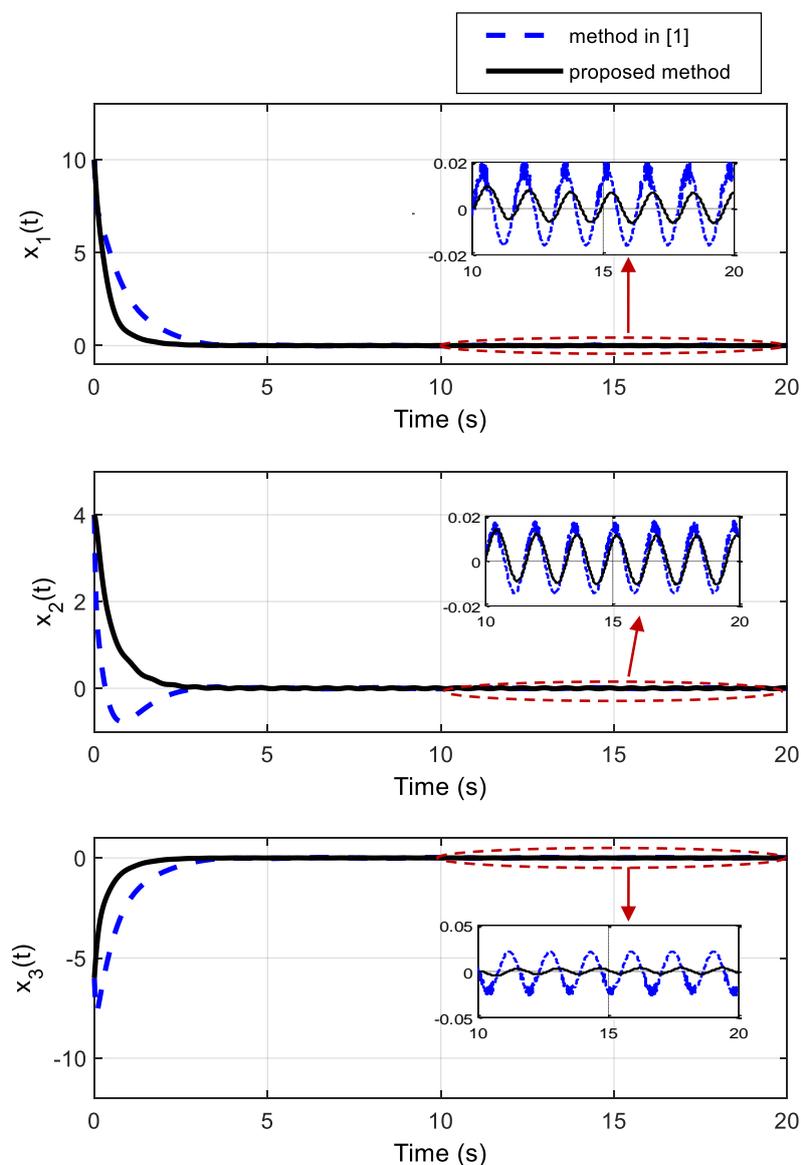


Figure 1. State trajectories of the unstable nonlinear system (showing the stabilization efficiency of the proposed method compared to the method of [1]).

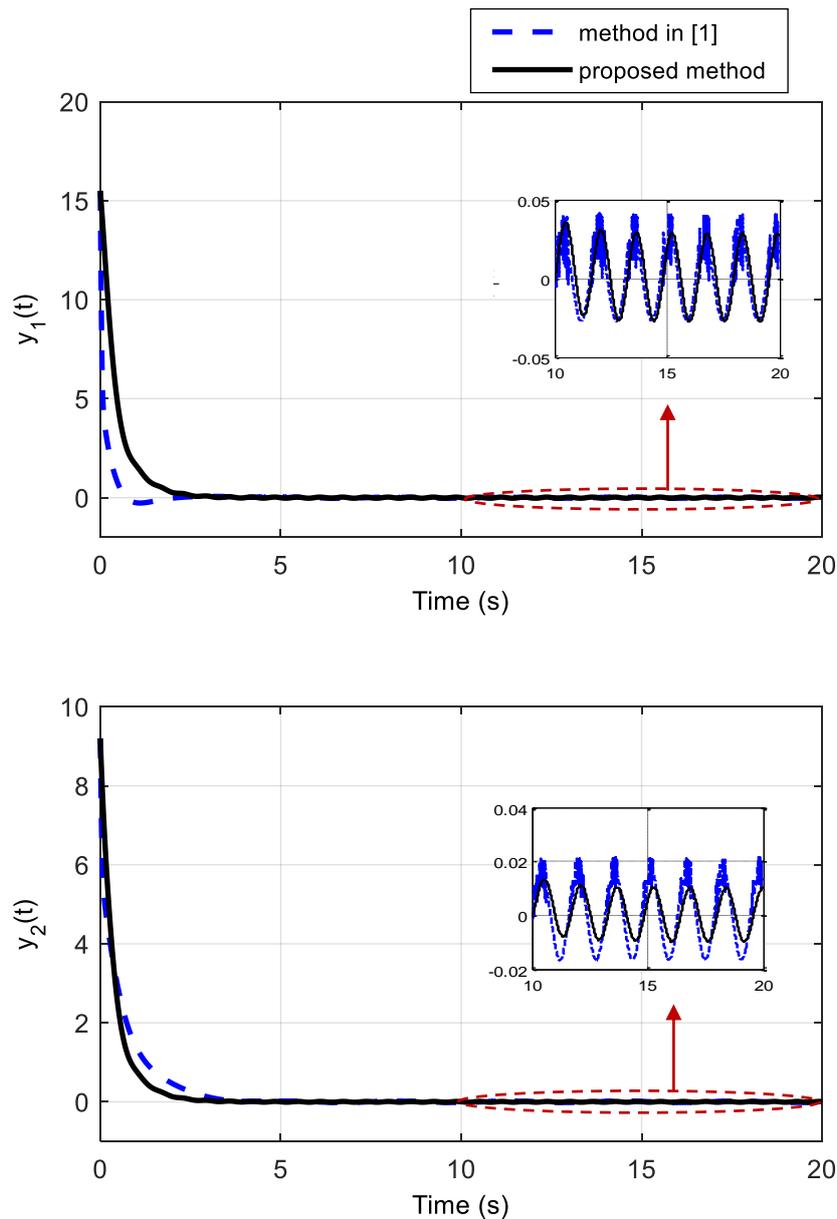


Figure 2. Time responses of the system outputs $y_1(t)$ and $y_2(t)$ (which show the better stabilization of the proposed method rather than the method of [1]).

Time responses of the outputs are presented in Figure 2, where it is indicated that the output response of [1] comprises high-frequency oscillations that are practically undesired. These simulation outcomes confirm that the suggested control scheme has superior performance in comparison with the controller of [1].

In what follows, the robustness of the proposed control technique is confirmed in a different situation. In this scenario, the new initial states are given as $x(0) = [-5 \ 8 \ 4]^T$. Moreover, the parametric uncertainties are considered 10 times larger. Time trajectories of the states and outputs are presented in Figures 3 and 4. These plots demonstrate the fast-transient responses of the recommended control scheme over the other method. The figures confirm the appropriate performance and robustness of the proposed technique in the new situation in comparison with the control method of [1].

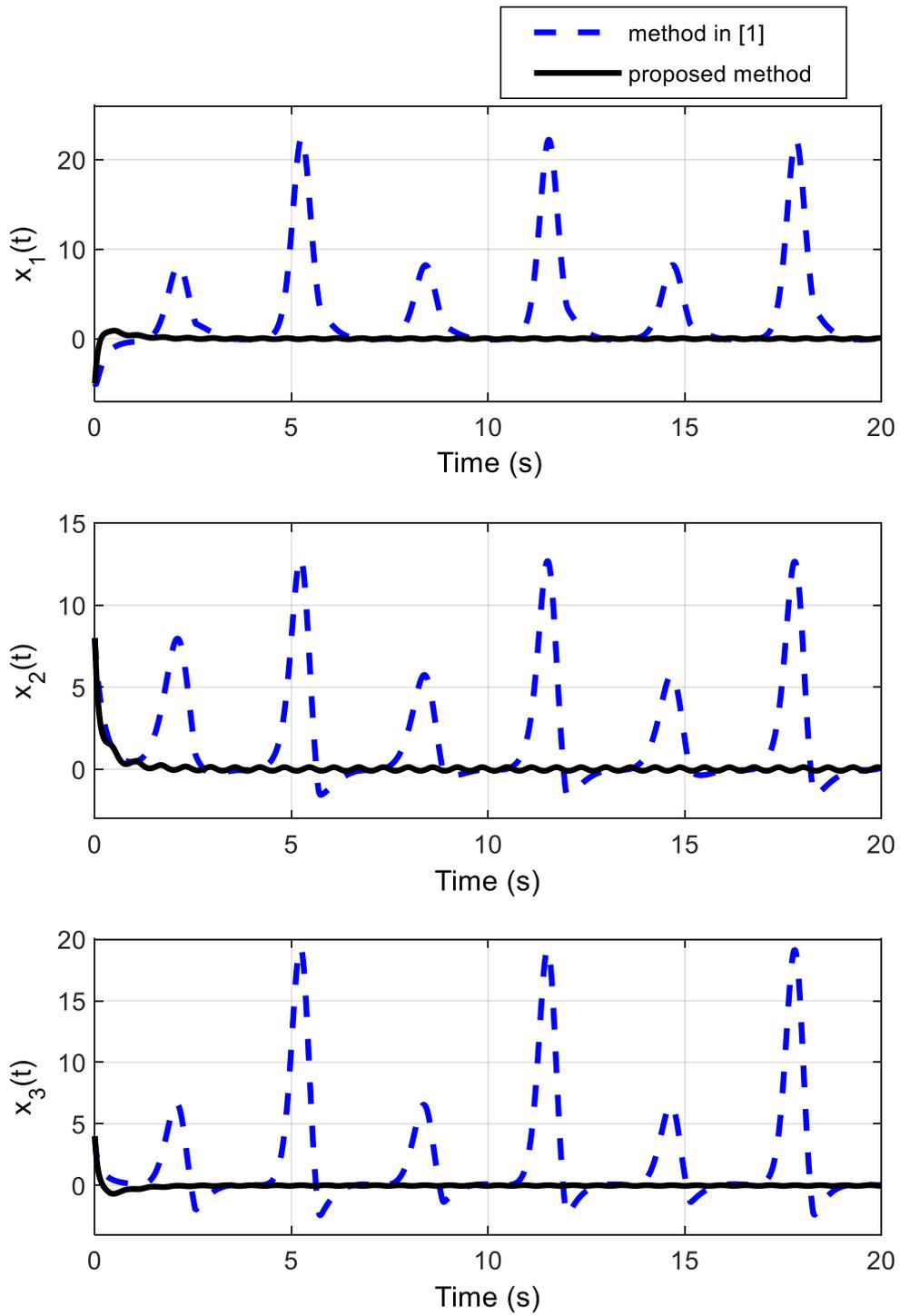


Figure 3. System states for the second scenario with different initial conditions and larger uncertainties (showing the better stabilization performance of the proposed method).

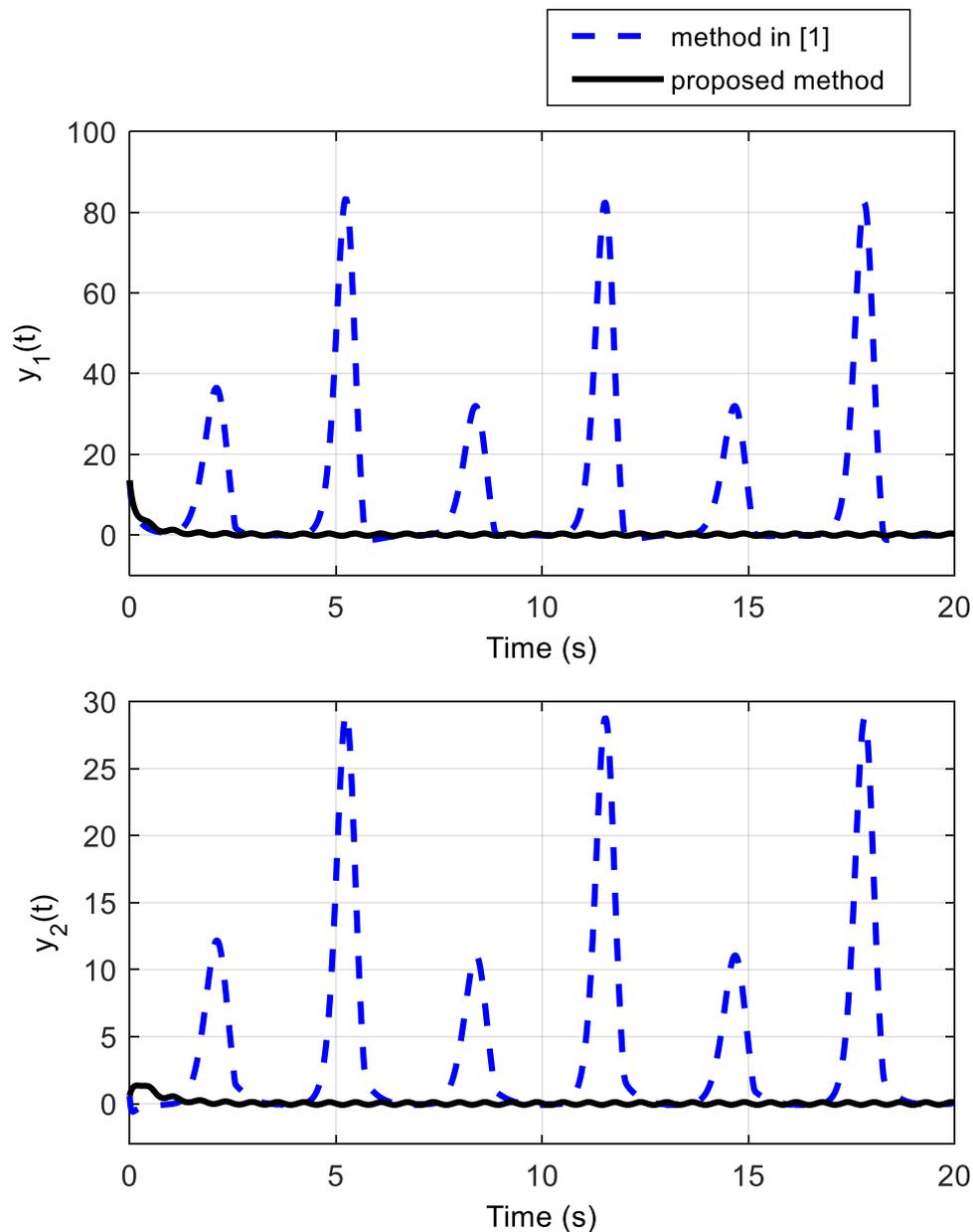


Figure 4. System outputs for the second scenario with different initial conditions and larger uncertainties (displaying the better stabilization performance of the proposed method compared to the method of [1]).

4.2. Example B: A Practical Rotational Inverted Pendulum

In the second example, we set out to balance a pendulum in an unstable upright equilibrium position in an RIP system, which is recognized as an appropriate test platform to assess the performance of various techniques. This under-actuated mechanical system has received the consideration of many investigators in various fields of study such as robotics, aerospace, marine vehicles, and pointing control. The RIP shown in Figure 5 is composed of a rotational servo-motor driving the output gear, rotational arm, and an inverted pendulum. Suppose α_p , θ_a , m_p , l_p , r_a , u , τ_a , and J_b represent the pendulum angle, arm angle, pendulum mass, pendulum length, arm length, control input,

motor torque, and moment of inertia of the effective mass, respectively. The dynamics of the RIP with constant time delay, friction, and backlash effects are specified by:

$$A_p \ddot{\theta}_a - (C_p \sin \alpha_p) \dot{\alpha}_p^2 + (B_p \sin 2\alpha_p) \dot{\alpha}_p \dot{\theta}_a + F_p \dot{\theta}_a + G_p \operatorname{sgn}(\dot{\theta}_a) + H_p \theta_a = I_p u + \frac{A_p B_p - C_p^2}{B_p} \begin{bmatrix} \alpha_p(t - \tau) \\ \dot{\alpha}_p(t - \tau) \\ \theta_a(t - \tau) \\ \dot{\theta}_a(t - \tau) \end{bmatrix}, \tag{9}$$

$$B_p \ddot{\alpha}_p - (B_p \sin \alpha_p \cos \alpha_p) \dot{\theta}_a^2 - D_p \sin \alpha_p + E_p \dot{\alpha}_p = 0 \tag{10}$$

where E_p , F_p , I_p , H_p , and G_p are pendulum damping coefficient, arm damping constant, controller signal coefficient, elasticity constant, and arm Coulomb friction, respectively. A_p , B_p , C_p , and D_p are considered as [38]:

$$\begin{aligned} A_p &= m_p r_a^2 + J_b, \\ B_p &= \frac{1}{3} m_p l_p^2, \\ C_p &= \frac{1}{2} m_p r_a l_p, \\ D_p &= \frac{1}{2} m_p g l_p. \end{aligned} \tag{11}$$

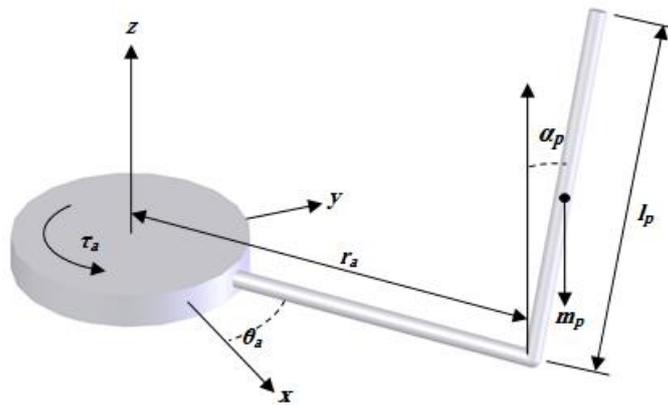


Figure 5. Schematic diagram of the RIP system (which illustrates that the connected motor to the arm causes the balancing control of the inverted pendulum).

The constants of the model (9) and (10) are set as:

$$A_p = 3.291, D_p = 6.052, G_p = 1.428, B_p = 0.237, E_p = 0.0132, H_p = 1.72, C_p = 0.237, F_p = 14.283, I_p = 6.38.$$

Therefore, the nonlinear time-delay model (9) and (10) can easily be written in the form of (1) as follows:

$$\begin{aligned} \dot{x} &= \begin{pmatrix} 0 \\ (\sin x_1 \cos x_1) x_4^2 + 25.54 \sin x_1 \\ 0 \\ (0.072 \sin x_1) x_2^2 - (0.072 \sin 2x_1) x_2 x_4 - 0.43 \operatorname{sgn}(x_4) \end{pmatrix} \\ &+ \begin{pmatrix} 0 & 1 & 0 & 0 \\ 0 & -0.056 & 0 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & -0.52 & -4.34 \end{pmatrix} x + 0.93 \begin{pmatrix} 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 1 & 1 & 1 & 1 \end{pmatrix} x(t - \tau) + \begin{pmatrix} 0.2 \\ 0.1 \\ 0.1 \\ 1.94 \end{pmatrix} u, \\ y &= (1 \ 0 \ 0 \ 0) x, \end{aligned} \tag{12}$$

where $x = [\alpha_p \ \dot{\alpha}_p \ \theta_a \ \dot{\theta}_a]^T$. The time delay and the initial states are considered as $x(0) = [\pi \ -1 \ -4 \ 2]^T$ and $\tau = 2$, respectively. Moreover, the Lipschitzian matrix is given by $L = \text{diag}(0.2, 0.3, 0.4, 0.8)$. Using YALMIP[®] solver, matrices H, P, P_1, F are obtained as:

$$H = \begin{bmatrix} 0.7213 & -0.1484 & 0.1765 & 1.7343 \\ -0.1484 & 0.7503 & 0.1916 & 1.9435 \\ 0.1765 & 0.1916 & 0.5853 & 2.2188 \\ 1.7343 & 1.9435 & 2.2188 & 12.6601 \end{bmatrix}, P = \begin{bmatrix} 1.1107 & 0.4934 & 0.4132 & 0.4817 \\ 0.4934 & 1.7864 & 1.1968 & -0.1674 \\ 0.4132 & 0.1968 & 1.3576 & 0.6499 \\ 0.4817 & -0.1674 & 0.6499 & 2.2046 \end{bmatrix}, P_1 = \begin{bmatrix} 7.8512 & 1.6463 & 9.5245 & 23.4526 \\ 1.6463 & 1.2598 & 2.0790 & 5.7018 \\ 9.5245 & 2.0790 & 12.1973 & 29.2457 \\ 23.4526 & 5.7018 & 29.2457 & 70.6682 \end{bmatrix}, F = [-3.5140 \ -3.2700 \ -3.3280 \ -2.0280].$$

The variation of the system state is illustrated in Figure 6. It is clear from this figure that the proposed approach is able to lead the system states to zero irrespective of the time delay and nonlinearities. Figure 7 indicates the required controller signal to make the states converge to zero. As a result, the robust performance of the proposed method is verified through these simulation results.

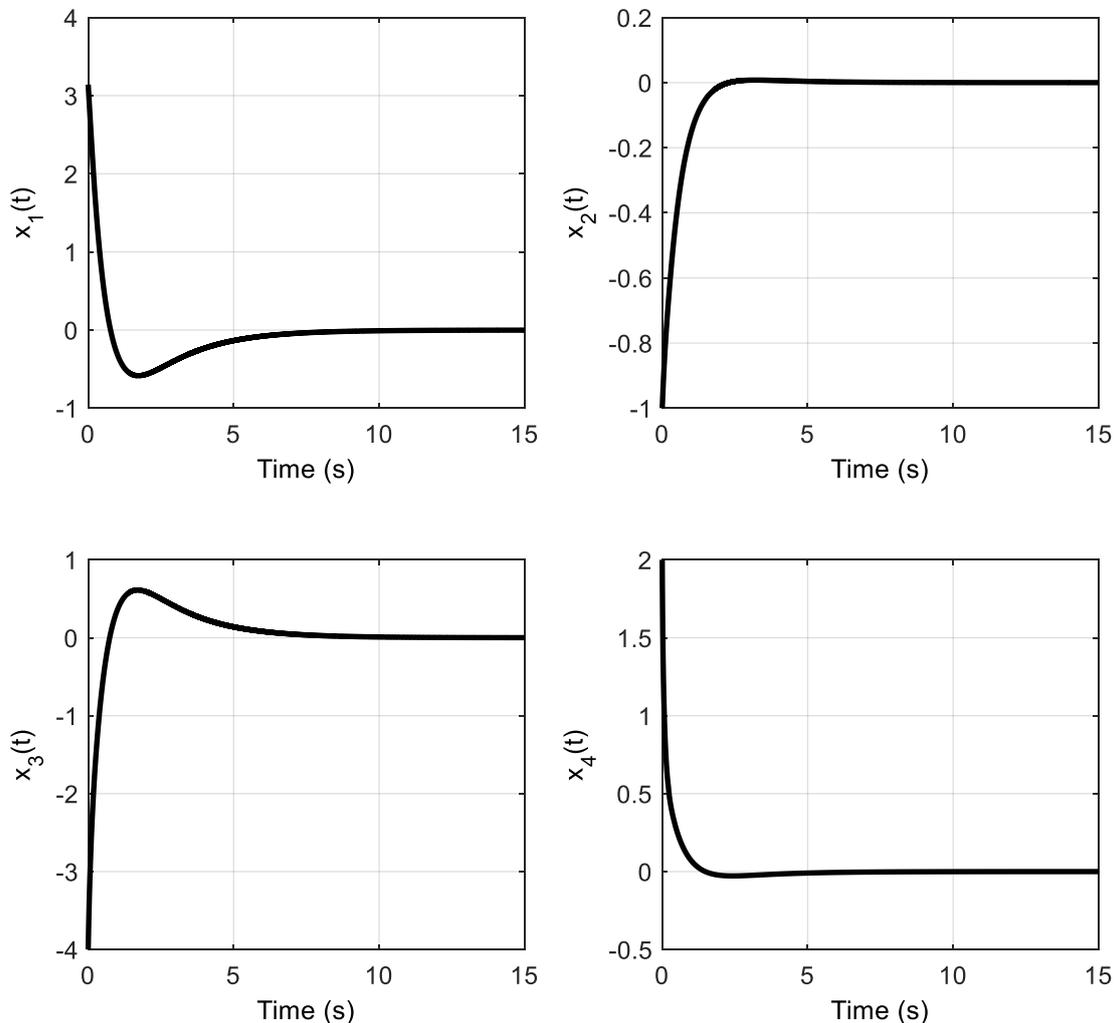


Figure 6. States of the RIP system (which shows that the angular position and velocity of the pendulum and arm are stabilized).

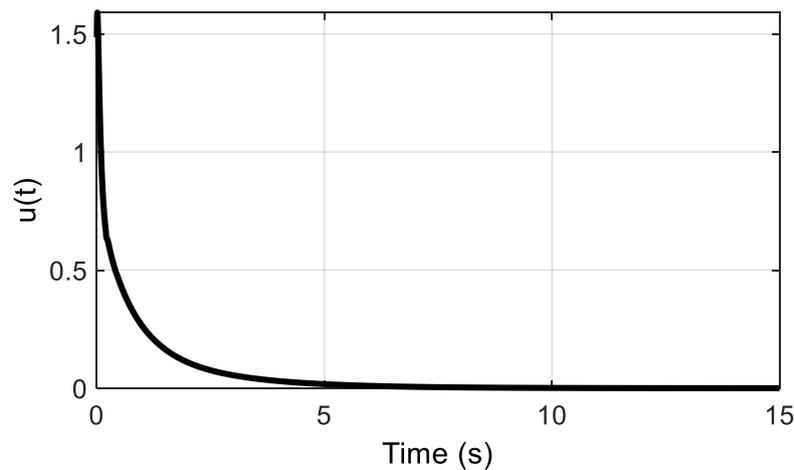


Figure 7. Control input (motor control signal $u(t)$).

Furthermore, we performed some experiments on the practical RIP that is constructed in the Department of Electrical Engineering of University of Zanjan, Zanjan, Iran. Figure 8 shows the experimental RIP system. The angular position of the pendulum was measured using the encoder E40S (Autonics Company, Busan, Republic of Korea). The practical results for the pendulum angle, arm angle, and controller input are presented in Figures 9 and 10. In the practical results, the time interval (0, 5.1) (swing-up period) is removed from the figures to focus on the control performance in the balance mode. The experimental results are similar to the simulations. This result illustrates that the offered control technique has a satisfactory and robust performance.



Figure 8. Experimental RIP system (the rotating arm stabilizes the inverted pendulum).

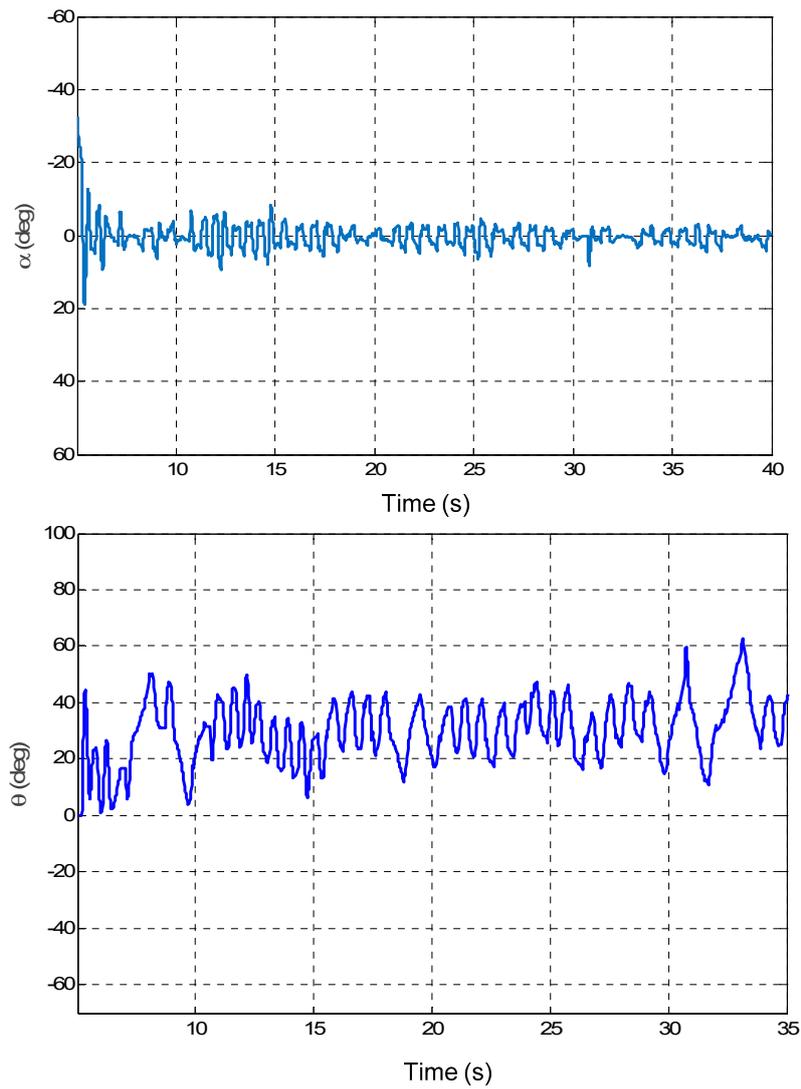


Figure 9. Pendulum angle (α) and arm angle (θ) of the experimental RIP system.

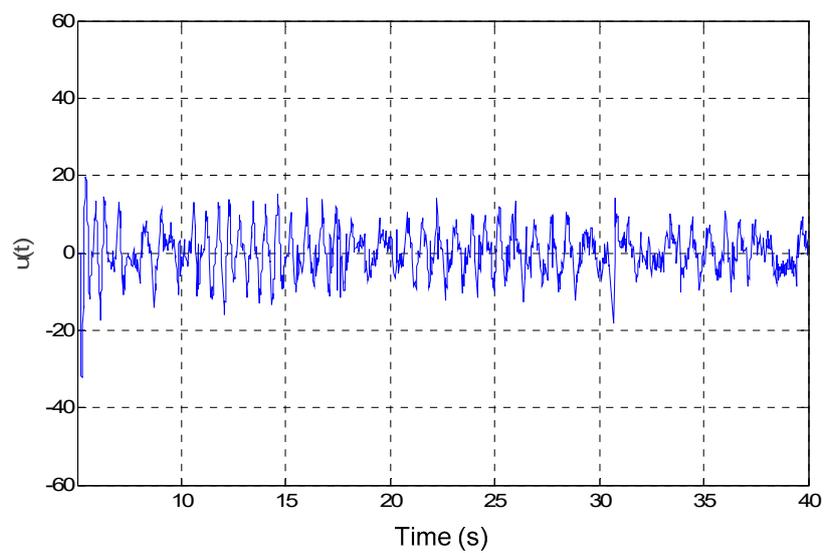


Figure 10. Control signal of the experimental RIP system ($u(t)$).

5. Conclusions

In this study, the problem of developing a nonlinear state-feedback controller was considered for the stabilization of time-delay nonlinear systems subject to structured uncertainties. Following the Lyapunov–Krasovskii stability theory, a nonlinear state-feedback control was derived to overcome the time delay and provide asymptotic stability of the states as well as robustness of the system against parametric uncertainties. In fact, the mentioned problem was converted into a common optimization problem that was then solved in the form of LMI. The gains of the control law were stated by the sufficient conditions via LMIs. The resultant LMI was comparatively straightforward in the computational aspect. Numerical simulation and practical outcomes were shown to approve the efficiency of the proposed approach and satisfactory outcomes were achieved. The suggested procedure can reach a promising tracking efficiency for high-order nonlinear perturbed systems. Moreover, the extension of the proposed control method based on gain-scheduled linear parameter-varying (LPV) control will be presented in future work.

Author Contributions: M.G. and S.M. were responsible for conceptualization, methodology, and software. S.M. were responsible for supervision and editing. S.S. and S.H.H. were responsible for formal analysis and writing—original draft preparation. All authors have read and agreed to the published version of the manuscript.

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Conflicts of Interest: The authors declare no conflict of interest.

Appendix A

If (5) is replaced into (1), we have:

$$\dot{x}(t) = f(x) - f(0) + (A + BF)x(t) + (\Delta A + \Delta BF)x(t) + A_1x(t - \tau) - \Delta BB^{-1}f(0). \quad (\text{A1})$$

Let us construct the following Lyapunov–Krasovskii functional:

$$V = x^T Px + \int_0^\tau x^T(t-s)P_1x(t-s)ds, \quad (\text{A2})$$

with $P > 0$ and $P_1 > 0$ as two real symmetric matrices that are obtained using LMI condition (7). The time-derivative of (A2) is derived as:

$$\dot{V} = \dot{x}^T Px + x^T P\dot{x} + x^T(t-\tau)P_1x(t-\tau) - x^T P_1x. \quad (\text{A3})$$

Substituting (A1) into (A3) gives:

$$\begin{aligned} \dot{V} = & (f(x) - f(0))^T Px + x^T(A^T + F^T B^T)Px + x^T(\Delta A^T + F^T \Delta B^T)Px + x^T(t-\tau)A_1^T Px \\ & - (\Delta BB^{-1}f(0))^T Px + x^T P(f(x) - f(0)) + x^T P(A + BF)x + x^T P(\Delta A + \Delta BF)x \\ & + x^T P A_1 x(t-\tau) - x^T P \Delta BB^{-1}f(0) + x^T(t-\tau)P_1x(t-\tau) - x^T P_1x, \end{aligned} \quad (\text{A4})$$

where the following special structures to the perturbations ΔA and ΔB are considered as [39]:

$$\begin{aligned} \Delta A &= D_A \Delta_A E_A \\ \Delta B &= D_B \Delta_B E_B \end{aligned} \quad (\text{A5})$$

where Δ_A and Δ_B are unknown real matrices. The idea behind condition (A5) is to capture all of the uncertainty in ΔA and ΔB and capture the structure of the perturbations using the other four matrices. Considering (4) and (A4) and using (A5), one obtains:

$$\begin{aligned} \dot{V} \leq & (f(x) - f(0))^T Px + x^T (A^T + F^T B^T) Px + x^T (E_A^T \Delta_A^T D_A^T + F^T E_B^T \Delta_B^T D_B^T) Px \\ & + x^T (t - \tau) A_1^T Px - (D_B \Delta_B E_B B^{-1} f(0))^T Px + x^T P (f(x) \\ & - f(0)) + x^T P (A + BF) x + x^T P (D_A \Delta_A E_A + D_B \Delta_B E_B F) x \\ & + x^T P A_1 x (t - \tau) - x^T P D_B \Delta_B E_B B^{-1} f(0) + x^T (t - \tau) P_1 x (t - \tau) \\ & - x^T P_1 x - (f(x) - f(0))^T I (f(x) - f(0)) + x^T L^T L x. \end{aligned} \tag{A6}$$

Equation (A6) can be rewritten as:

$$\begin{aligned} \dot{V} \leq & x^T ((A^T + F^T B^T) P + P(A + BF) + E_A^T \Delta_A^T D_A^T P + P D_A \Delta_A E_A + F^T E_B^T \Delta_B^T D_B^T P \\ & + P D_B \Delta_B E_B F - P_1 + L^T L) x + x^T (t - \tau) P_1 x (t - \tau) - (f(x) \\ & - f(0))^T I (f(x) - f(0)) + (f(x) - f(0))^T P x + x^T P (f(x) \\ & - f(0)) - (D_B \Delta_B E_B B^{-1} f(0))^T P x - x^T P D_B \Delta_B E_B B^{-1} f(0) + x^T (t \\ & - \tau) A_1^T P x + x^T P A_1 x (t - \tau) \end{aligned} \tag{A7}$$

where, using Lemma 1, it implies that:

$$\begin{aligned} \dot{V} \leq & x^T ((A^T + F^T B^T) P + P(A + BF) + \gamma_A E_A^T \Delta_A^T \Delta_A E_A + \frac{1}{\gamma_A} P D_A D_A^T P \\ & + \gamma_B F^T E_B^T \Delta_B^T \Delta_B E_B F + \frac{1}{\gamma_B} P D_B D_B^T P - P_1 + L^T L) x + x^T (t \\ & - \tau) P_1 x (t - \tau) - (f(x) - f(0))^T I (f(x) - f(0)) + (f(x) \\ & - f(0))^T P x + x^T P (f(x) - f(0)) \\ & + \gamma_C f^T(0) B^{-T} E_B^T \Delta_B^T \Delta_B E_B B^{-1} f(0) + \frac{1}{\gamma_C} x^T P D_B D_B^T P x + x^T (t \\ & - \tau) A_1^T P x + x^T P A_1 x (t - \tau) \end{aligned} \tag{A8}$$

Now, if the conditions $\Delta_A^T \Delta_A \leq I$ and $\Delta_B^T \Delta_B \leq I$ are satisfied, then, we obtain from (A8) that:

$$\begin{aligned} \dot{V} \leq & x^T ((A^T + F^T B^T) P + P(A + BF) + \gamma_A E_A^T E_A + \frac{1}{\gamma_A} P D_A D_A^T P + \gamma_B F^T E_B^T E_B F \\ & + \frac{1}{\gamma_B} P D_B D_B^T P - P_1 + L^T L) x + x^T (t - \tau) P_1 x (t - \tau) - (f(x) \\ & - f(0))^T I (f(x) - f(0)) + (f(x) - f(0))^T P x + x^T P (f(x) \\ & - f(0)) + \gamma_C f^T(0) B^{-T} E_B^T E_B B^{-1} f(0) + \frac{1}{\gamma_C} x^T P D_B D_B^T P x + x^T (t \\ & - \tau) A_1^T P x + x^T P A_1 x (t - \tau) \end{aligned} \tag{A9}$$

which, further, can be written as:

$$\dot{V} \leq \Psi^T \Gamma \Psi, \tag{A10}$$

where

$$\Psi = [x^T x^T (t - \tau) (f(x) - f(0))^T f^T(0)]^T, \tag{A11}$$

$$\Gamma = \begin{bmatrix} K_1 & P A_1 & P & 0 \\ * & P_1 & 0 & 0 \\ * & * & -I & 0 \\ * & * & * & \gamma_C B^{-T} E_B^T E_B B^{-1} \end{bmatrix} < 0, \tag{A12}$$

where $K_1 = (A^T + F^T B^T) P + P(A + BF) - P_1 + L^T L + \gamma_A E_A^T E_A + \frac{1}{\gamma_A} P D_A D_A^T P + \gamma_B F^T E_B^T E_B F + (\frac{1}{\gamma_B} + \frac{1}{\gamma_C}) P D_B D_B^T P$.

Applying the Schur complement in (A12) yields:

$$\Gamma = \begin{bmatrix} K_2 & PA_1 & P & 0 & PD_A \\ * & P_1 & 0 & 0 & 0 \\ * & * & -I & 0 & 0 \\ * & * & * & \gamma_C B^{-T} E_B^T E_B B^{-1} & 0 \\ * & * & * & * & -\gamma_A I \end{bmatrix} < 0, \quad (\text{A13})$$

where $K_2 = (A^T + F^T B^T)P + P(A + BF) - P_1 + L^T L + \gamma_A E_A^T E_A + \gamma_B F^T E_B^T E_B F + (\frac{1}{\gamma_B} + \frac{1}{\gamma_C})PD_B D_B^T P$. Using the Schur complement lemma, inequality (A13) is written as:

$$\Gamma = \begin{bmatrix} K_3 & PA_1 & P & 0 & PD_A & PD_B \\ * & P_1 & 0 & 0 & 0 & 0 \\ * & * & -I & 0 & 0 & 0 \\ * & * & * & \gamma_C B^{-T} E_B^T E_B B^{-1} & 0 & 0 \\ * & * & * & * & -\gamma_A I & 0 \\ * & * & * & * & * & -\frac{\gamma_B \gamma_C}{\gamma_B + \gamma_C} \end{bmatrix} < 0, \quad (\text{A14})$$

where $K_3 = (A^T + F^T B^T)P + P(A + BF) - P_1 + L^T L + \gamma_A E_A^T E_A + \gamma_B F^T E_B^T E_B F$.

In what follows, let γ_A , γ_B , and γ_C be fixed. Using a congruence transformation as $T = \text{diag}(P^{-1}, P^{-1}, I, I, I, I)$ and pre-post multiplying T in (A14), we have:

$$\begin{bmatrix} K_4 & A_1 P^{-1} & I & 0 & D_A & D_B \\ P^{-1} A_1 & P^{-1} P_1 P^{-1} & 0 & 0 & 0 & 0 \\ I & 0 & -I & 0 & 0 & 0 \\ 0 & 0 & 0 & \gamma_C B^{-T} E_B^T E_B B^{-1} & 0 & 0 \\ D_A^T & 0 & 0 & 0 & -\gamma_A I & 0 \\ D_B^T & 0 & 0 & 0 & 0 & -\frac{\gamma_B \gamma_C}{\gamma_B + \gamma_C} \end{bmatrix} < 0, \quad (\text{A15})$$

where $K_4 = P^{-1} A^T + P^{-1} F^T B^T + AP^{-1} + BFP^{-1} + \gamma_B P^{-1} F^T E_B^T E_B FP^{-1} - P^{-1} P_1 P^{-1} + P^{-1}(L^T L + \gamma_A E_A^T E_A)P^{-1}$. Now, based on the transformations $Q = P^{-1}, Y = FP^{-1}, H = P^{-1} P_1 P^{-1}, M^{-1} = L^T L + \gamma_A E_A^T E_A$, one obtains:

$$\begin{bmatrix} K_4 & A_1 Q & I & 0 & D_A & D_B \\ QA_1 & H & 0 & 0 & 0 & 0 \\ I & 0 & -I & 0 & 0 & 0 \\ 0 & 0 & 0 & \gamma_C B^{-T} E_B^T E_B B^{-1} & 0 & 0 \\ D_A^T & 0 & 0 & 0 & -\gamma_A I & 0 \\ D_B^T & 0 & 0 & 0 & 0 & -\frac{\gamma_B \gamma_C}{\gamma_B + \gamma_C} \end{bmatrix} < 0, \quad (\text{A16})$$

where $K_4 = QA^T + Y^T B^T + AQ + BY + \gamma_B Y^T E_B^T E_B Y - H + QM^{-1}Q$. Finally, if the Schur complement is applied to (A16), LMI (7) is obtained.

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